Minimization of Machine Speed Deviation by Eigenvalue Adjusment of State Matrix of Interconnected Network Fitted with UPFC and Power Oscillation Damper

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ABSTRACT: In this paper the approach of eigenvalue adjustment of state matrix of a large interconnected network is used to minimize the machine speed deviations for a large interconnected Network fitted with FACTS device and Power Oscillation Damper. As a foundational work the generalized mathematic model of multi-machine power system with embedded FACTS was developed. This is principally aimed at providing benchmark models to enable in-depth analysis of dynamic and transient performances of the Machines when subjected to credible disturbance scenarios at different operating points. The results obtained clearly reveals the effectiveness of this approach

Keywords: UPFC, POD, Damping, Speed Deviation

I. INTRODUCTION

Wide variations in consumer demands often lead to perturbations in the power system dynamics. Whenever there is an imbalance between the mechanical input power to the rotor and the electrical output power from the stator electromechanical oscillation is triggered in the generator, the power transmission network consists of transmission lines, transformers and other energy storing accessories through which the ripples of this energy imbalance is transmitted as oscillation across the whole network. These oscillations are sometimes prolonged low frequency types that can lead to adverse effects on the performance of the power system in general and in extreme cases, culminate in catastrophic system failure. This research intends to deploy eigenvalue analysis method to a network embedded with FACTS Device and Power Oscillation Damper for the purpose of damping these oscillations and maintain the performance of an integrated power system at optimal level.

LITERATURE REVIEW

Eigenvalue analysis (modal analysis) is a valuable tool in analysis of power system small signal stability. It provides information about the inherent dynamic characteristic of the power system and assists in its design. It is typically used in studies of inter-area oscillations [1]. Eigenvalues can be used to determine the small-signal stability of the power systems. They are also good indicators for the bifurcation points such as saddle-node bifurcation and Hopf bifurcation. Eigenvalues and eigenvectors can be used to determine mode shapes, participation factors, controllability and observability of each mode.

Eigenvalues with the largest real parts and eigenvalues with the least damping ratios are of great importance for power system applications. In power system analysis, eigenvalues are widely used to study system uncertainty, estimate stability boundary, and enhance system stability performance. Eigenvalue computation is a linear analysis technique based on the linearized model. In reference [2], the author summarize the main research developments in the computational methods for eigenvalue problems during the 20th century.

In general, the group of critical eigenvalues is usually a small subset in the whole spectrum most of the early research was on accelerating the computation of these eigenvalues especially for large systems. Several dominant sub-spectrum algorithms exist to serve this purpose such as the power method, the inverse power method, subspace iteration, and the Arnoldi method [3], etc. The efficiencies of dominant eigenvalue calculation methods are discussed and compared in references [3] and [4]. The matrix transformation for the dominant eigenvalues is described in reference [5]. Among all of the dominant eigenvalue computation algorithms, the Arnoldi method is believed to be the most efficient approach. But the method is heavily influenced by the selection of the number of guard vectors. The only way to explore this is by numerical tests.

It has been assumed that the dominant eigenvalue is the critical eigenvalue that will eventually cross the imaginary axis [6], but this may not always be the case especially when the current operating point is some distance away from the instability boundary.

III. DESCRIPTION OF TEST SYSTEM

Figures 1 and 2 shows the single line diagram of the test systems used. Details of system data are given in [7-8]. The sub-transient models of the systems synchronous machines were used with the detailed dynamic model of other embedded system components.

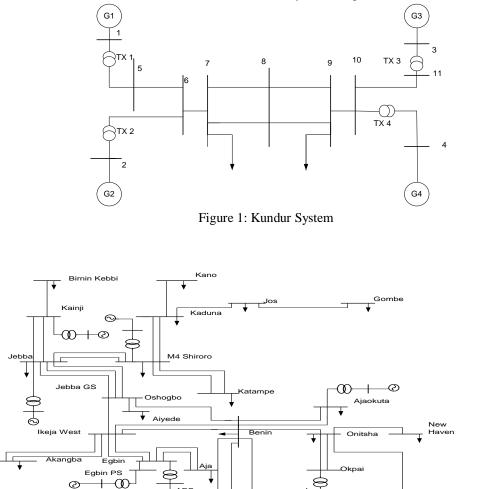
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IV. PROPOSED APPROACH

An electric power system can be represented by a set of non-linear differential-algebraic equations (DAE), as follows [9]

$$\dot{x} = f(x, y, u)$$

$$0 = g(x, y, u)$$

$$w = h(x, y, u)$$
(1)

(3)

After solving the power flow problem, a modal analysis is carried out by computing the eigenvalues and the participation factors of the state matrix of the system [2]. Linearizing equation (1) at an equilibrium point yields equation (2)

$$\begin{bmatrix} \Delta \dot{x} \\ 0 \\ \Delta w \end{bmatrix} = \begin{bmatrix} F_x & F_y & F_u \\ G_x & G_y & G_u \\ H_x & H_y & H_u \end{bmatrix} \begin{bmatrix} \Delta x \\ \Delta y \\ \Delta u \end{bmatrix}$$
(2)
$$F_x = \nabla_x^F f, F_y = \nabla_y^F f, F_u = \nabla_u^F f, G_x = \nabla_x^G g, G_y = \nabla_y^G g, G_u = \nabla_u^G g, H_x = \nabla_x^H h, H_y = \nabla_y^H h,$$
$$H_u = \nabla_u^H h$$

Eliminating Δy , and assuming that power flow Jacobian G_{y} is non-singular (i.e. the system does

not show a singularity induced bifurcation), the state matrix A ($A \in \mathcal{R}^{nxn}$) of the system is given

by $A = F_x - F_y G_y^{-1} G_x$ and the state space representation of equation (2) is

$$\Delta \dot{x} = A\Delta x + B\Delta u$$

$$\Delta w = C\Delta x + D\Delta u$$

$$B = C \Delta x + D\Delta u$$

where the input matrix $B = F_u - F_y G_y^{-1} G_u$ ($B \in \mathcal{R}^{nxr}$), the output matrix $C = H_x - H_y G_y^{-1} G_x$ ($C \in \mathcal{R}^{mxn}$), and the feed forward matrix $D = F_u - H_y G_y^{-1} G_u$ ($D \in \mathcal{R}^{mxr}$). The state-space representation is the base for controller design [10].

 $x \in \mathcal{R}^n$, is the vector of the state variables

 $u \in \mathcal{R}^l$ is a set of controllable parameters (e.g controller reference signal)

 $y \in \mathcal{R}^m$ is the vector of algebraic variables (e.g., bus voltage magnitudes and phase angles);

 $w \in \mathcal{R}^k$ is a set of output variables (e.g., line current flows);

f is a set of differential equations that represents system and controller dynamics:

$$(f: \mathcal{R}^n. \mathcal{R}^m. \mathcal{R}^l \to \mathcal{R}^n)$$

g is a set of algebraic equations that represents the transmission network power flows $(g: \mathcal{R}^n, \mathcal{R}^m, \mathcal{R}^l \to \mathcal{R}^m)$

h is a set of equations that represents output variables (e.g., measurements), such as line power flows and rotor angle speeds. ($h: \mathcal{R}^n, \mathcal{R}^m, \mathcal{R}^l \to \mathcal{R}^k$)

The small signal stability analysis of equation (2) is based on the eigenvalues of the state matrix A. If the state matrix, A, for a typical power system having n distinct eigenvalues [11] then Λ , Φ and Ψ are: the diagonal matrix of eigenvalues, matrices of right and left eigenvectors given, respectively by compact matrix expression of equation (4)

$$A\Phi = \Phi\Lambda \Longrightarrow \Lambda = \Phi^{-1}A\Phi$$

$$\Psi A = \Lambda \Psi \Longrightarrow \Lambda = \Psi A \Psi^{-1}$$

$$\Phi \triangleq \Psi^{-1}$$
(4)

Where $\Lambda \triangleq \text{Diag}[\lambda_1, \lambda_2, ..., \lambda_i, ..., \lambda_n]$ and $\lambda_i \triangleq \sigma_i \pm j\omega_i$ being i^{th} eigenvalue of the A-matrix; $\Phi \triangleq [\phi_1, \phi_2, ..., \phi_i, ..., \phi_n]; \Psi \triangleq [\psi_1^T, \psi_2^T, ..., \psi_i^T, ..., \psi_n^T]^T; \phi_i \in \mathcal{R}^{nx1} \& \psi_i \in \mathcal{R}^{1xn}$

Complex eigenvalues occur in conjugate pairs (since the state matrix is real), and each pair correspond to an oscillatory mode. The real component of the eigenvalues gives the damping, and the imaginary component gives the frequency of oscillation. A negative real part represents a damped oscillation whereas a positive real part represents oscillation of increasing amplitude. For the complex i^{th} pair of eigenvalues ($\sigma_i \pm j\omega_i$) the damping ratio ζ_i is given by:

$$\zeta_i = \frac{-\sigma_i}{\sqrt{\sigma_i^2 + \omega_i^2}} \tag{5}$$

It can also be shown [12] that $\lambda_i = \zeta_i \omega_{ni} \pm j \omega_{ni} \sqrt{1 - \zeta_i^2}$ where ω_{ni} is the undamped natural frequency associated with the *i*th eigenvalues. The poorly damped oscillatory modes usually have small values of damping ratios and the selected control structures must act on them to improve overall dynamic stability the system of interest. Modal analysis as applied to linearized system is revisited in sequel.

V. RESULTS AND DISCUSSIONS

A. Result of Modal Analysis Test System 1

The Modal analysis is used in identifying the group of generators that constitute an area. Table 2 shows the list of left eigenvectors for the speed components of the system eigenvalues, Figure 3 shows the compass plot of the right eigenvectors of the speed state components of the 4 generators in the system. The plot is an indicator of the machines oscillating coherently in an area for the selected poor damped inter-area mode 3 in Table 1.

S/No	EIGENVALUE	DAMPING RATIO	EIGENVALUE WITH	DAMPINGRATIO
	WITHOUT POD	WITHOUT POD	POD	WITH POD
1	-0.8752 + 6.2808i	0.1380	-0.8628 + 6.2597i	0.1365
2	-0.9760 + 6.5082i	0.1483	-0.9572 + 6.4659i	0.1464
3	-0.2428 + 3.7434i	0.0647	-0.7690 + 3.1067i	0.2630

Table 2: Right Eigenvectors of Speed State Component

Generators	Left Eigenvectors
1	0.0621 - 0.0256i
2	0.3749 - 0.4097i
3	-0.0227 - 0.1096i
4	-0.4352 + 0.2327i

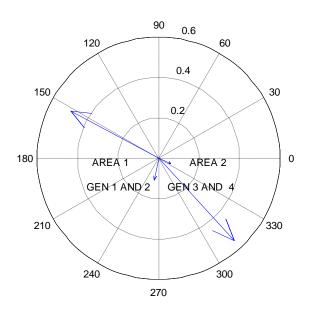


Figure 3: Mode Shape of Kundur two Area System

B. Result of Dynamic Simulation

Figures 4 and 5 shows the result of dynamic simulation with three phase fault on Bus 11 for 0.05 sec. The effect of the damping on machine deviation from 1 pu without and with Power Oscillation Damper is clearly revealed.

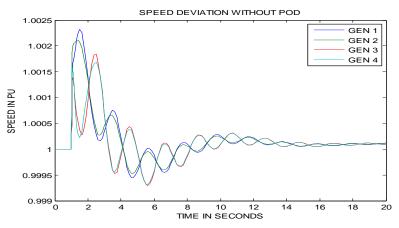


Figure 4: Speed Deviation of the Four Machines without POD

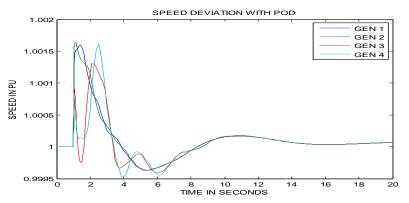


Figure 5: Speed Deviation of the Four Machines With POD

C. Result of Modal Analysis Test System 2

The Modal analysis is used in identifying the group of generators that constitute an area. Figure 6 shows the compass plot of the right eigenvectors of the speed state components of the 11 generators in the system. The plot is an indicator of the machines oscillating coherently in an area for the selected poor damped inter-area mode 1 in Table 3.

Table 3: System	n Eigenvalues	and Dampng Ratios

	EIGENVALUES	DAMPING	EIGENVALUES	DAMPING
	WITHOUT POD	RATIO	WITH POD RATI	0
1	-1.6407 + 8.4932i	0.0219	-1.7438 + 8.5469i	0.0229
2	2 -0.5812 + 8.1009i	0.0088	-0.5544 + 8.0712i	0.0085
3	3 -1.0184 + 8.2293i	0.0148	-0.9708 + 8.0572i	0.0147
4	-1.0199 + 7.9320i	0.0159	-0.9688 + 7.7068i	0.0161
5	5 -0.6920 + 6.3196i	0.0171	-0.6600 + 7.1578i	0.0128
6	5 -0.9109 + 6.1541i	0.0235	-0.7063 + 6.2629i	0.0178
7	- 0.4904 + 5.6519i	0.0152	-0.4256 + 5.3115i	0.0150
8	3 -3.4159 + 4.0681i	0.1211	-3.3244 + 3.9953i	0.1231

9 -0.3174 + 3.3260i	0.0284	-0.7221 + 3.6100i 0.0533	
10 -1.6348 + 2.0122i	0.2432	-1.5902 + 2.0403i 0.2376	

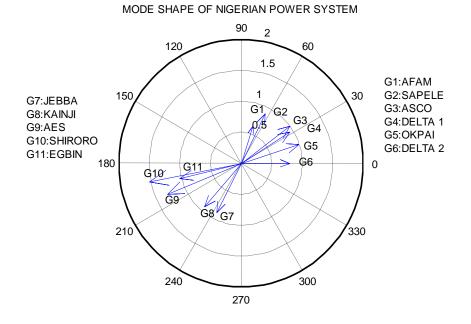


Figure 6: Mode shape of Nigerian Power System

D. Result of Dynamic Simulation of Nigerian Power System

The Dynamic Simulation was performed with and without POD. Figures 7 and 8 shows the speed deviation of the 11 generators from 1 pu without and with POD.

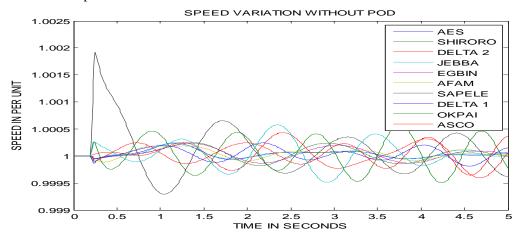


Figure 7: Speed Deviation of the 11 Generators in the System without POD

SPEED VARIATION WITH POD

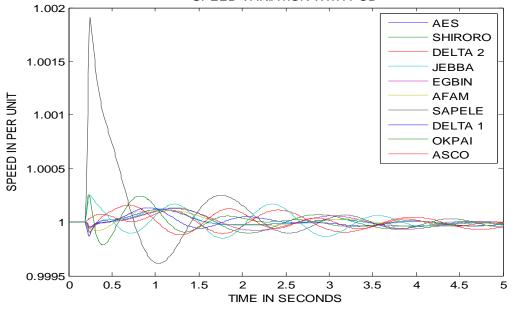


Figure 8: Speed Deviations of the 11 Generators in the System with POD

VII. CONCLUSION

In this work the Nigerian national grid is embedded UPFC controller that is equipped with Power Oscillation Damper, this constitutes the main case study system. The methods of Eigenvalue analysis, modal analysis and dynamic simulations, were deployed to show the impact of this damping method on Machine speed deviations. The results obtained are clear indications of the effectiveness of this proposed method.

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